

## **Deformation of continuous media**

# Plan

- 1 Strain field measurements
- 2 Material placement
- 3 Deformation gradient
  - Deformation of lines, surface and volumes
  - Polar decomposition
  - Simple extension and simple glide
  - Strain measures
  - Summary
- 4 Velocity gradient tensor
  - Strain rate tensor
  - Spin tensor
  - Example: simple glide, single vortex
  - Summary
- 5 Stresses
  - Principle of virtual power
  - Nominal and Piola–Kirchhoff stress tensor

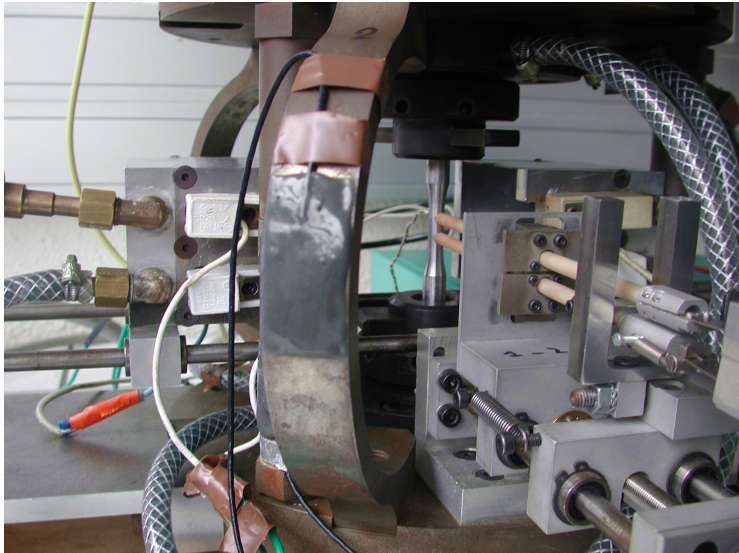
# Extensometry

measuring the relative **displacement** of two material points

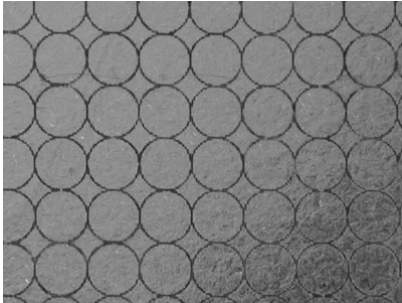


# Extensometry

measuring the relative **displacement** of two material points



# Strain field measurements



# Strain field measurements

tensile test



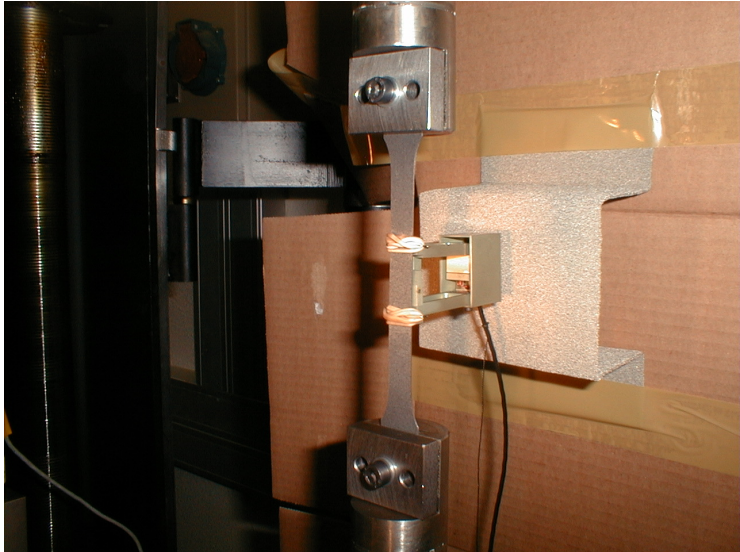
# Strain field measurements

tensile test



# Strain field measurements

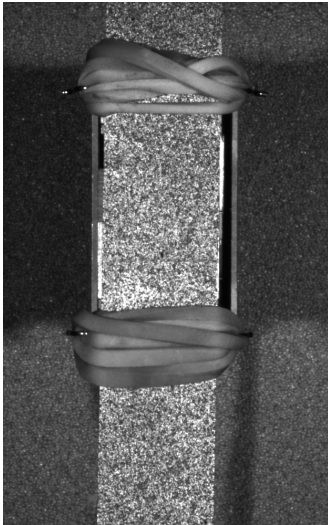
tensile test



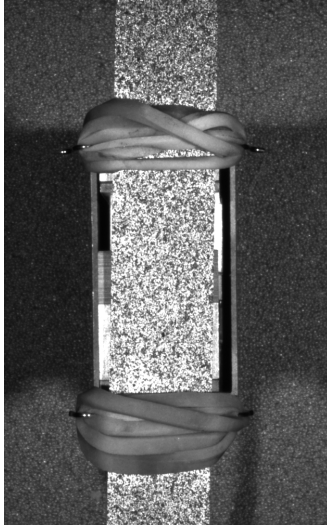


# Strain field measurements

elongation of the gauge length



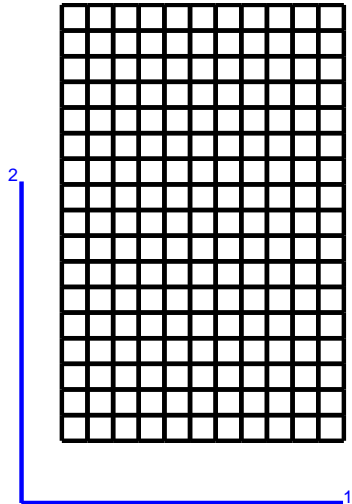
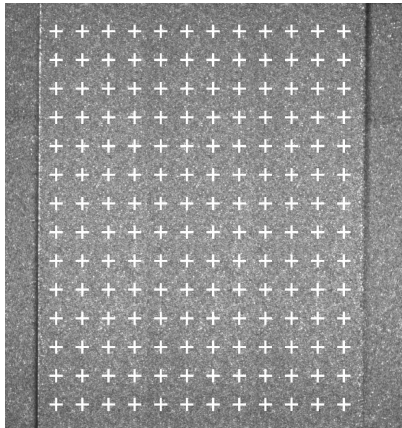
initial state



deformed state

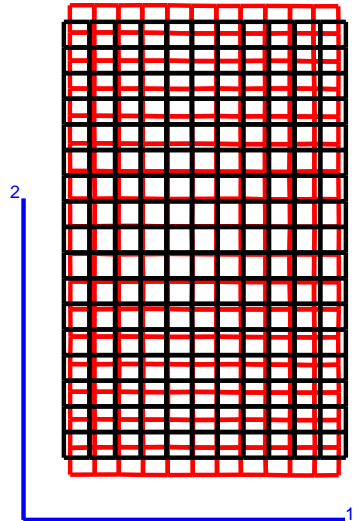
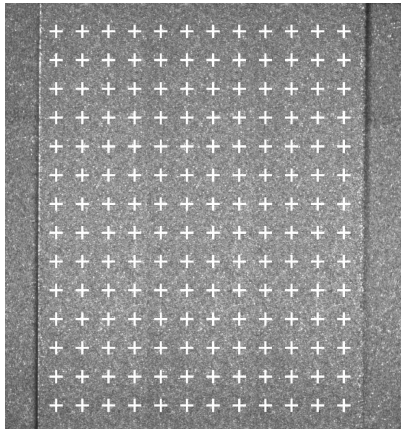
# Strain field measurements

image correlation technique : locating patterns around a grid of material points during deformation



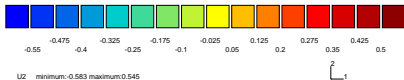
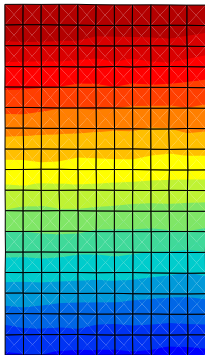
# Strain field measurements

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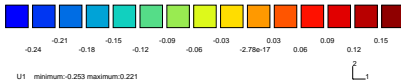
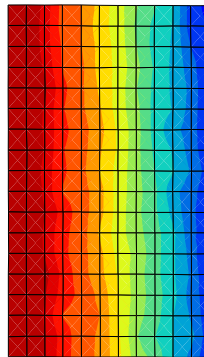


# Strain field measurements

field of ...



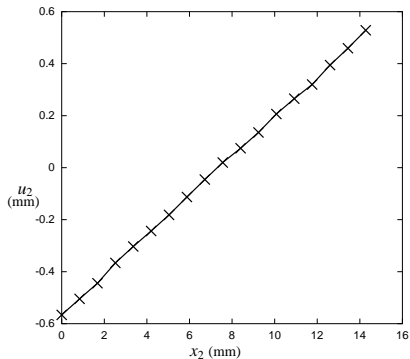
displacement  $u_2$



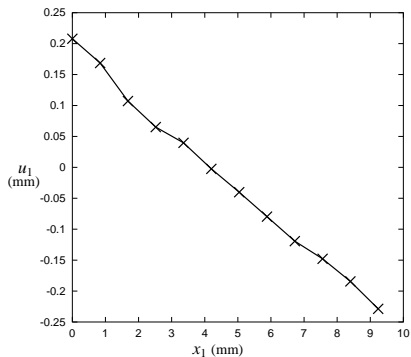
displacement  $u_1$  in mm

# Strain field measurements

displacement of points along a line



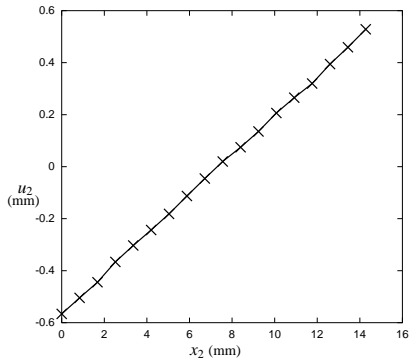
vertical



horizontal

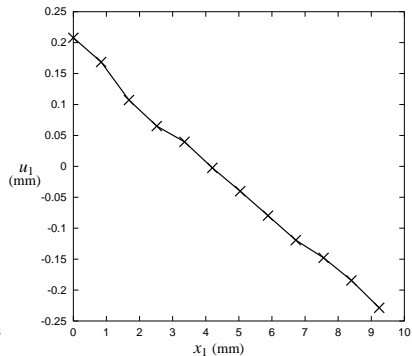
# Strain field measurements

displacement of points along a line



vertical

$$F_{22} - 1 = \frac{\partial u_2}{\partial x_2}$$



horizontal

$$F_{11} - 1 = \frac{\partial u_1}{\partial x_1}$$

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# Notations

Euclidean tensor fields; orthonormal basis ( $\underline{\mathbf{e}}_1, \underline{\mathbf{e}}_2, \underline{\mathbf{e}}_3$ )

- zero<sup>th</sup> order tensors : scalar field  $f(\underline{\mathbf{X}}, t)$
- first order tensors : vectors  $\underline{\mathbf{x}}(\underline{\mathbf{X}}, t)$

$$\underline{\mathbf{x}} = x_i \underline{\mathbf{e}}_i, \quad [\underline{\mathbf{x}}] = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

- second order tensors : linear mapping / bilinear forms  $\underline{\underline{\mathbf{C}}}(\underline{\mathbf{X}}, t)$

$$\underline{\underline{\mathbf{C}}} = C_{ij} \underline{\mathbf{e}}_i \otimes \underline{\mathbf{e}}_j, \quad [\underline{\underline{\mathbf{C}}}] = \begin{bmatrix} C_{11} & C_{12} & C_{13} \\ C_{21} & C_{22} & C_{23} \\ C_{31} & C_{32} & C_{33} \end{bmatrix}$$

- operations between tensors with respect to an **orthonormal** basis

$$\underline{\mathbf{a}} \cdot \underline{\mathbf{b}} = a_i b_i = [\underline{\mathbf{a}}]^T [\underline{\mathbf{b}}], \quad \underline{\underline{\sigma}} \cdot \underline{\mathbf{n}} = \sigma_{ij} n_j \underline{\mathbf{e}}_i, \quad [\underline{\underline{\sigma}} \cdot \underline{\mathbf{n}}] = [\underline{\underline{\sigma}}] [\underline{\mathbf{n}}]$$

$$\underline{\mathbf{m}} \cdot \underline{\underline{\sigma}} \cdot \underline{\mathbf{n}} = m_i \sigma_{ij} n_j = [\underline{\mathbf{m}}]^T [\underline{\underline{\sigma}}] [\underline{\mathbf{n}}]$$

$$\underline{\underline{\sigma}} : \underline{\underline{\mathbf{L}}} = \sigma_{ij} L_{ij} = \text{trace}(\underline{\underline{\sigma}} \cdot \underline{\underline{\mathbf{L}}}^T) = \text{trace}([\underline{\underline{\sigma}}][\underline{\underline{\mathbf{L}}}]^T)$$



# Volume element $dV$ of continuum mechanics

- Notion of **material point** : An infinitesimal volume  $dV$  around  $\mathbf{X}$
- $dV \sim$  Representative Volume Element

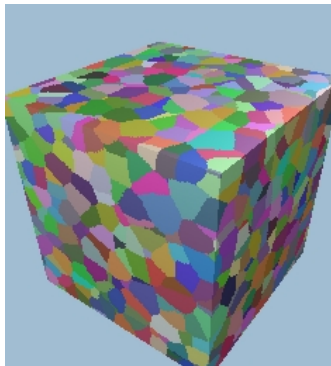
$$d \ll L_{VER} \ll L$$

$d$  size of heterogeneities

$L_{RVE}$  size of RVE

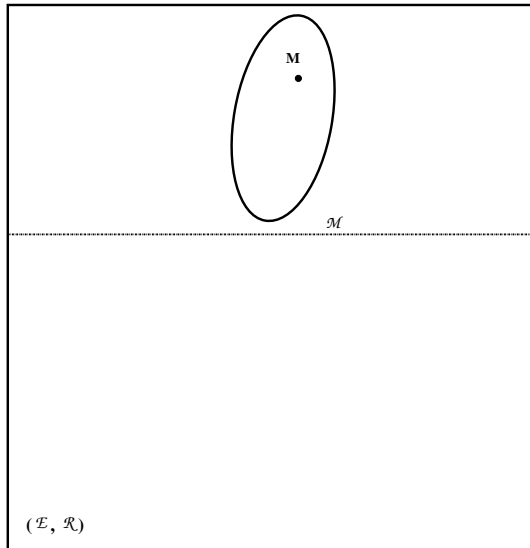
$L$  structural size

- we follow the material point without considering the particles inside the RVE...



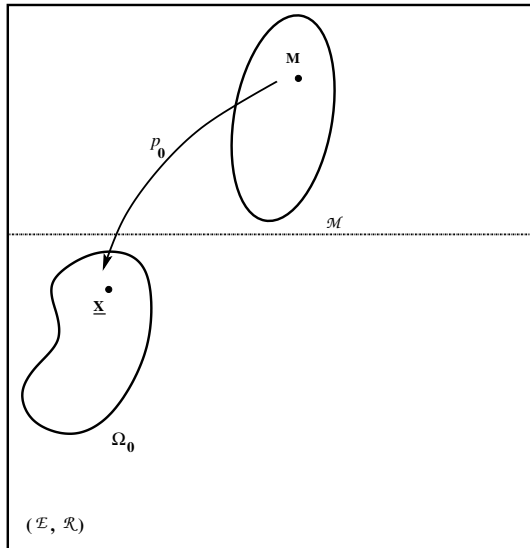
RVE for a metal  
polycrystal

# Material body $\mathcal{M}$



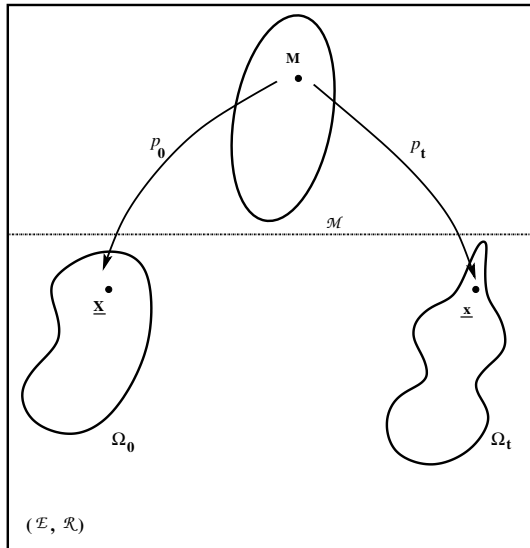
Material body  $\mathcal{M}$   
is a set of material  
points  $M$

# Reference placement in physical space



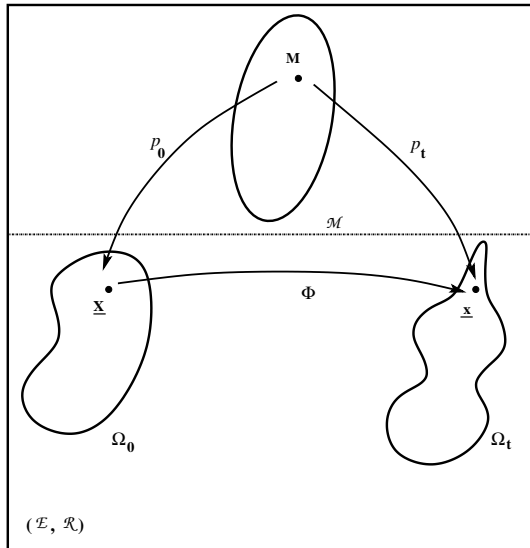
- $\Omega_0$  is the configuration of  $\mathcal{M}$  in physical space  $\mathcal{E}$  with respect to the observer  $\mathcal{R}$  at time  $t_0$
- material point  $M \in \mathcal{M}$  occupies place  $\underline{X}$  in this configuration
- we choose it as the **reference configuration**

# Current configuration in physical space



- $\Omega_t$  is the configuration of material body  $\mathcal{M}$  in physical space  $\mathcal{E}$  at time  $t$
- material point  $M \in \mathcal{M}$  occupies place  $\underline{x}$  in this configuration
- we call it **current configuration**

# Current configuration in physical space



- The motion

$$\underline{x} = \Phi(\underline{X}, t)$$

links  $\Omega_0$  to  $\Omega_t$

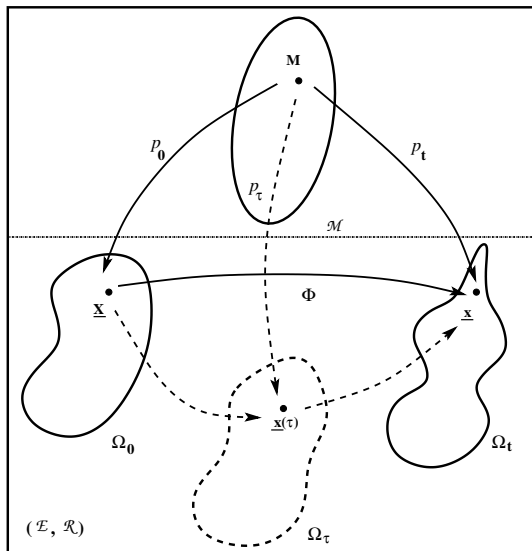
- The motion is **bijective and bi-continuous**

$$\underline{X} = \Phi^{-1}(\underline{x}, t)$$

★ no fission!

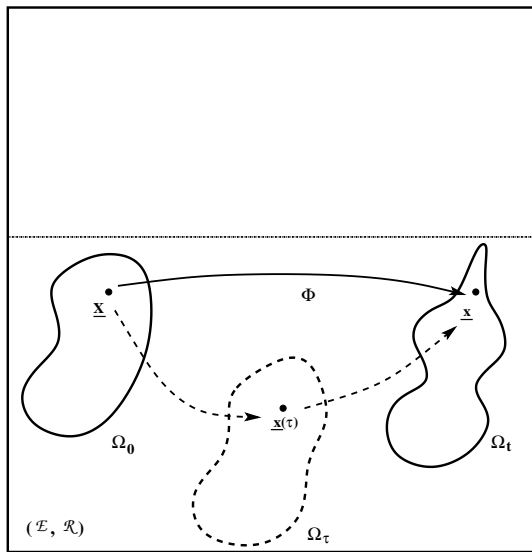
★ no fusion!

# Intermediate configuration of the material body



- The configuration fo  $\mathcal{M}$  at  $0 \leq \tau \leq t$  is denoted  $\Omega_\tau$  and called **intermediate configuration** of the material body
- The motion  $\Phi(\underline{X}, \tau)_{0 \leq \tau \leq t}$  records **the deformation history** of the material body

# Deformation of continuous media



- The motion

$$\underline{x} = \Phi(\underline{X}, t)$$

links  $\Omega_0$  to  $\Omega_t$

- **Displacement** of material point is

$$\underline{u}(\underline{X}, t) = \underline{x} - \underline{X}$$

$$\underline{u}(\underline{X}, t) = \Phi(\underline{X}, t) - \underline{X}$$

# Lagrangian vs. Eulerian approaches

- materials with an underlying **microstructure** : generally “solids”

The volume element  $dV \in \Omega_0$  around  $\underline{\mathbf{X}}$  becomes  $dv \in \Omega_t$  around  $\underline{\mathbf{x}}$ .  $dV$  and  $dv$  contain the same particles.

field of tensor function  $F(\underline{\mathbf{X}}, t)$

Lagrangian approach

- materials without any underlying **microstructure** : generally “fluids”

Particles can be interchanged, they are not labelled. One is concerned with the mean velocity of particles going through  $dv$  around the geometrical point  $\underline{\mathbf{x}} \in \Omega_t$  at time  $t$

field of tensor function  $f(\underline{\mathbf{x}}, t)$

Eulerian approach

- Les points de vue lagrangien et eulériens sont équivalents :

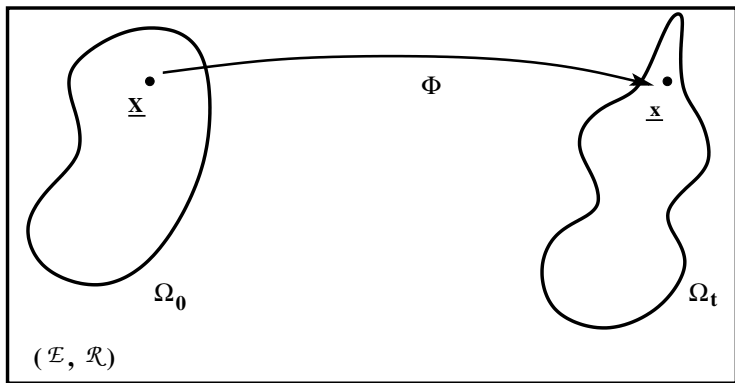
$$f(\underline{\mathbf{x}}, t) := F(\Phi^{-1}(\underline{\mathbf{x}}, t), t), \quad F(\underline{\mathbf{X}}, t) := f(\Phi(\underline{\mathbf{X}}, t), t)$$



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# Deformation of a continuum medium



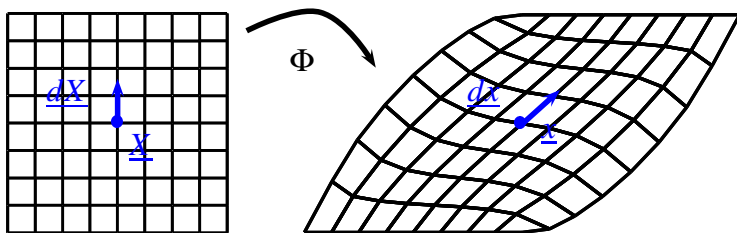
motion  $\Phi$

$$\underline{x} = \Phi(\underline{X}, t)$$

displacement field

$$\underline{u}(\underline{X}, t) = \underline{x} - \underline{X} = \Phi(\underline{X}, t) - \underline{X}$$

# Deformation gradient



tangent linear mapping associated with  $\Phi$

$$\Phi(\underline{\mathbf{X}} + \underline{\mathbf{dX}}) - \Phi(\underline{\mathbf{X}}) = \frac{\partial \Phi}{\partial \underline{\mathbf{X}}} \cdot \underline{\mathbf{dX}} + o(\underline{\mathbf{X}}, \underline{\mathbf{dX}})$$

material line element  $\underline{\mathbf{dX}}$  initial and  $\underline{\mathbf{dx}}$  current

$$\underline{\mathbf{F}}(\underline{\mathbf{X}}, t) = \frac{\partial \Phi}{\partial \underline{\mathbf{X}}} = \text{Grad } \Phi = \underline{\mathbf{1}} + \text{Grad } \underline{\mathbf{u}}, \quad \underline{\mathbf{dx}} = \underline{\mathbf{F}} \cdot \underline{\mathbf{dX}}$$

The deformation gradient  $\underline{\mathbf{F}}$  is a local characterization of the motion  
 $(\underline{\mathbf{F}}(\underline{\mathbf{X}}, t = 0) = \underline{\mathbf{1}})$

## Deformation gradient

with respect to an orthonormal basis  $(\underline{\mathbf{e}}_i)_{i=1,3}$

$$dx_i = F_{ij}dX_j, \quad \text{avec} \quad F_{ij} = \frac{\partial x_i}{\partial X_j} = \delta_{ij} + \frac{\partial u_i}{\partial X_j} \quad \text{et} \quad \underline{\mathbf{F}} = F_{ij} \underline{\mathbf{e}}_i \otimes \underline{\mathbf{e}}_j$$

$$dx_1 = \frac{\partial \Phi_1}{\partial X_1} dX_1 + \frac{\partial \Phi_1}{\partial X_2} dX_2 + \frac{\partial \Phi_1}{\partial X_3} dX_3$$

$$dx_2 = \frac{\partial \Phi_2}{\partial X_1} dX_1 + \frac{\partial \Phi_2}{\partial X_2} dX_2 + \frac{\partial \Phi_2}{\partial X_3} dX_3$$

$$dx_3 = \frac{\partial \Phi_3}{\partial X_1} dX_1 + \frac{\partial \Phi_3}{\partial X_2} dX_2 + \frac{\partial \Phi_3}{\partial X_3} dX_3$$

$$\begin{bmatrix} dx_1 \\ dx_2 \\ dx_3 \end{bmatrix} = \begin{bmatrix} F_{11} & F_{12} & F_{13} \\ F_{21} & F_{22} & F_{23} \\ F_{31} & F_{32} & F_{33} \end{bmatrix} \begin{bmatrix} dX_1 \\ dX_2 \\ dX_3 \end{bmatrix}$$

the components of  $\underline{\mathbf{F}}$  are dimensionless

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# Transformation of a material volume element

- volume element: initial  $dV$  and current  $dv$

$$dV = \underline{\mathbf{dX}}_1 \cdot (\underline{\mathbf{dX}}_2 \wedge \underline{\mathbf{dX}}_3) = [\underline{\mathbf{dX}}_1, \underline{\mathbf{dX}}_2, \underline{\mathbf{dX}}_3] = \det(\underline{\mathbf{dX}}_1, \underline{\mathbf{dX}}_2, \underline{\mathbf{dX}}_3)$$

$$dv = [\underline{\mathbf{dx}}_1, \underline{\mathbf{dx}}_2, \underline{\mathbf{dx}}_3] = [\tilde{\mathbf{F}} \cdot \underline{\mathbf{dX}}_1, \tilde{\mathbf{F}} \cdot \underline{\mathbf{dX}}_2, \tilde{\mathbf{F}} \cdot \underline{\mathbf{dX}}_3]$$

$$dv = J dV$$

$$J = \det \tilde{\mathbf{F}} > 0$$

**Jacobian** of deformation

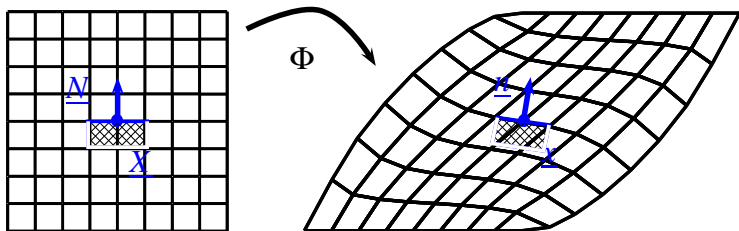
- the motion is **isochoric** at a point or at all points if  $J = 1$   
a material is **incompressible** if it can undergo only isochoric motions

# Mass conservation

$$\rho \, dv = \rho_0 \, dV = \rho \, J \, dV \implies \rho_0 = J \rho$$
$$\int_{\mathcal{D}(t)} \rho(\underline{\mathbf{x}}, t) \, dv = \int_{\mathcal{D}_0} \underbrace{\rho(\Phi(\underline{\mathbf{X}}, t), t)}_{\rho_0(\underline{\mathbf{X}})} J \, dV$$

with  $\mathcal{D}_0 = \Phi^{-1}(\mathcal{D}(t))$

## Transformation of a material surface element



$$\underline{dS} = \underline{dX}_1 \wedge \underline{dX}_3 = dS \underline{N}, \quad \underline{ds} = \underline{dx}_1 \wedge \underline{dx}_3 = ds \underline{n}$$

the surface element is defined by orthogonal material directions  $\underline{dX}_1$  and  $\underline{dX}_3$ . The surface element vector  $\underline{dS}$  does NOT transform like a line element:

$$\underline{ds} = J \tilde{\mathbf{F}}^{-T} . \underline{dS}$$

$\underline{dS}$  et  $\underline{ds}$  (resp.  $\underline{N}$  and  $\underline{n}$ ) are not made of the same material points in the initial and current configurations



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## Polar decomposition of deformation gradient

For all invertible  $\mathbf{F}$ , there exist two unique symmetric positive definite tensors  $\mathbf{U}$  et  $\mathbf{V}$  and a unique orthogonal tensor  $\mathbf{R}$  such that

$$\mathbf{F} = \mathbf{R} \cdot \mathbf{U} = \mathbf{V} \cdot \mathbf{R}$$

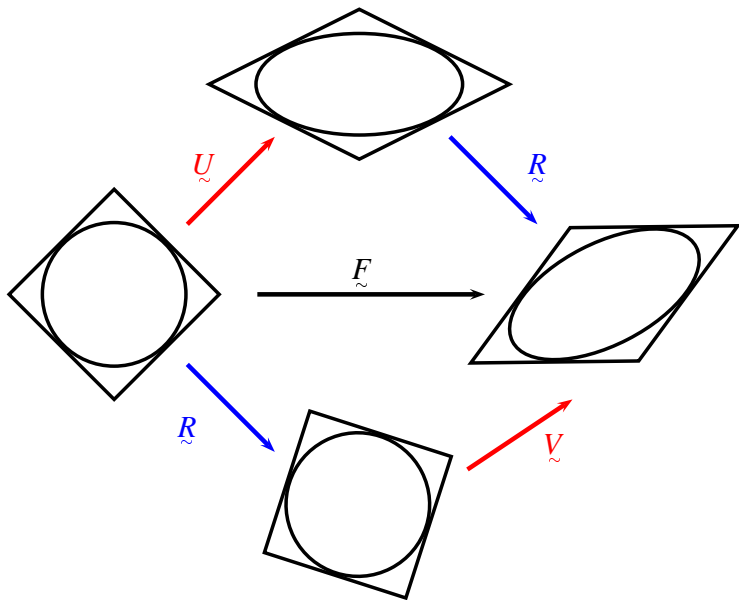
If  $\det \mathbf{F} > 0$ ,  $\mathbf{R}$  is a rotation (i.e.  $\det \mathbf{R} = +1$ ).

$\mathbf{R}$  polar rotation (3 degrees of freedom)

$$\mathbf{R}^{-1} = \mathbf{R}^T$$

$\mathbf{U}, \mathbf{V}$  stretch tensors (6 degrees of freedom)      $\mathbf{U}^T = \mathbf{U}, \quad \mathbf{V}^T = \mathbf{V}$

# Polar decomposition of deformation gradient



# Transformation of a principal triad of $\underline{\mathbf{U}}$

- **spectral decomposition** of  $\underline{\mathbf{U}}$  et  $\underline{\mathbf{V}}$

$$\underline{\mathbf{U}} \cdot \underline{\mathbf{V}}_r = \lambda_r \underline{\mathbf{V}}_r, \quad \lambda_r > 0 \quad (\text{no sum}), \quad \underline{\mathbf{U}} = \sum_{r=1}^3 \lambda_r \underline{\mathbf{V}}_r \otimes \underline{\mathbf{V}}_r$$

the eigen vectors are called **principal directions** or **principal axes** of  $\underline{\mathbf{U}}$ , and eigen values are called **principal stretches**

$$\underline{\mathbf{v}}_r = \underline{\mathbf{R}} \cdot \underline{\mathbf{V}}_r, \quad \underline{\mathbf{V}} = \sum_{r=1}^3 \lambda_r \underline{\mathbf{v}}_r \otimes \underline{\mathbf{v}}_r$$

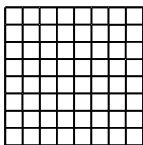
- A principal triad  $\underline{\mathbf{U}}$  transforms into an orthogonal triad. The orientation of the deformed triad with respect to the initial triad is exactly given by polar rotation  $\underline{\mathbf{R}}$

# Plan

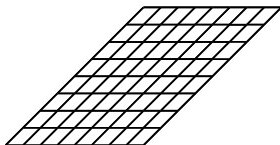
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# Homogeneous deformation

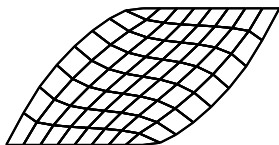
- homogeneous deformation :  $\underline{\mathbf{F}}(\underline{\mathbf{X}}, t) = \underline{\mathbf{F}}(t)$



initial



homogeneous



heterogeneous

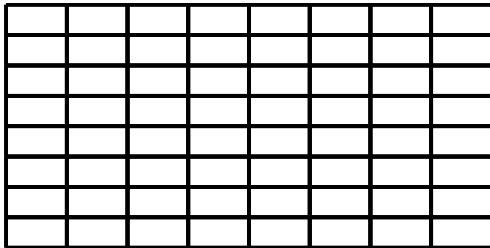
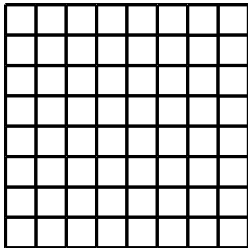
- corresponding motion / displacement field:

$$\underline{\mathbf{x}}(t) = \underline{\mathbf{F}}(t) \cdot \underline{\mathbf{X}} + \underline{\mathbf{c}}(t)$$

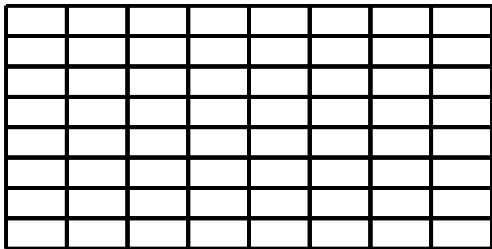
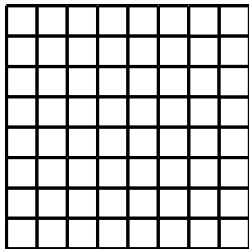
for any pair of material points (extensometry)

$$\underline{\mathbf{x}}_1 - \underline{\mathbf{x}}_2 = \underline{\mathbf{F}} \cdot (\underline{\mathbf{X}}_1 - \underline{\mathbf{X}}_2)$$

## Pure extension



## Pure extension



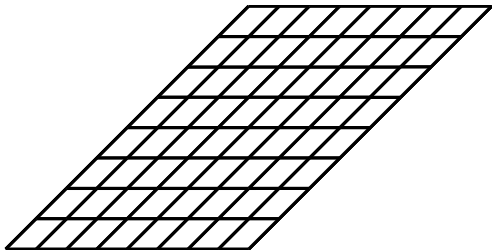
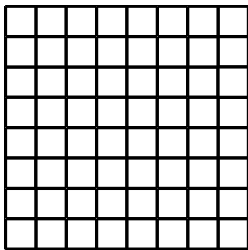
$$\begin{cases} x_1 = X_1(1 + \lambda) \\ x_2 = X_2 \\ x_3 = X_3 \end{cases}$$

$$\mathbf{\tilde{F}} = \mathbf{\tilde{1}} + \lambda \mathbf{e}_1 \otimes \mathbf{e}_1, \quad [\mathbf{\tilde{F}}] = \begin{bmatrix} 1 + \lambda & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

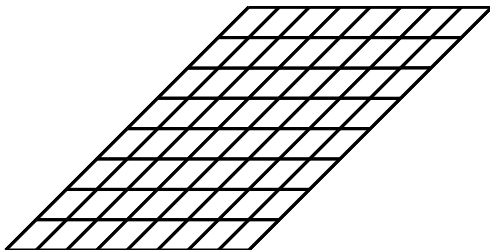
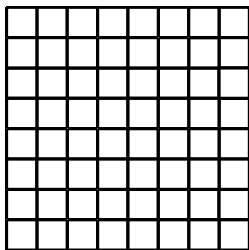
$$\mathbf{\tilde{R}} = \mathbf{\tilde{1}}, \quad \mathbf{\tilde{U}} = \mathbf{\tilde{F}}$$



## Simple glide



## Simple glide



$$\begin{cases} x_1 = X_1 + \gamma X_2 \\ x_2 = X_2 \\ x_3 = X_3 \end{cases}, \quad \mathbf{\tilde{F}} = \mathbf{\tilde{1}} + \gamma \mathbf{e}_1 \otimes \mathbf{e}_2, \quad [\mathbf{\tilde{F}}] = \begin{bmatrix} 1 & \gamma & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

## Simple glide

$$\tilde{\mathbf{u}} = \begin{bmatrix} \frac{1}{\sqrt{1 + (\gamma/2)^2}} & \frac{\gamma}{2\sqrt{1 + (\gamma/2)^2}} & 0 \\ \frac{\gamma}{2\sqrt{1 + (\gamma/2)^2}} & \frac{1 + \gamma^2/2}{\sqrt{1 + (\gamma/2)^2}} & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\tilde{\mathbf{v}} = \begin{bmatrix} \frac{1 + \gamma^2/2}{\sqrt{1 + (\gamma/2)^2}} & \frac{\gamma}{2\sqrt{1 + (\gamma/2)^2}} & 0 \\ \frac{\gamma}{2\sqrt{1 + (\gamma/2)^2}} & \frac{1}{\sqrt{1 + (\gamma/2)^2}} & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\tilde{\mathbf{R}} = \begin{bmatrix} \frac{1}{\sqrt{1 + (\gamma/2)^2}} & \frac{\gamma}{2\sqrt{1 + (\gamma/2)^2}} & 0 \\ \frac{-\gamma}{2\sqrt{1 + (\gamma/2)^2}} & \frac{1}{\sqrt{1 + (\gamma/2)^2}} & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

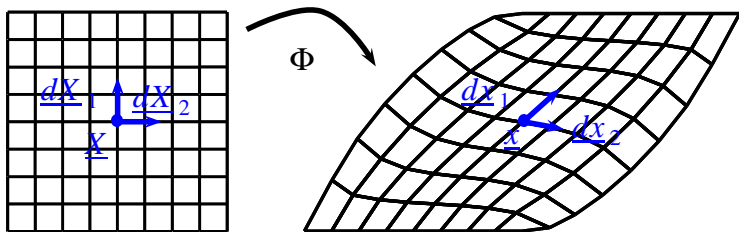
the polar rotation is a rotation with respect to axis  $\mathbf{e}_3$  and angle

$$\tan \theta = -\frac{\gamma}{2}$$

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## Cauchy–Green tensors



$$\underline{dx}_1 \cdot \underline{dx}_2 = (\underline{F} \cdot \underline{dX}_1) \cdot (\underline{F} \cdot \underline{dX}_2) = \underline{dX}_1 \cdot \underline{F}^T \cdot \underline{F} \cdot \underline{dX}_2 = \underline{dX}_1 \cdot \underline{C} \cdot \underline{dX}_2$$

right Cauchy–Green tensor  $\underline{C} = \underline{F}^T \cdot \underline{F}$  induces a metric on  $\Omega_0$

$$\underline{dX}_1 \cdot \underline{dX}_2 = \underline{dx}_1 \cdot \underline{B}^{-1} \cdot \underline{dx}_2$$

left Cauchy–Green tensor  $\underline{B} = \underline{F} \cdot \underline{F}^T$  induces a metric on  $\Omega_t$

( $\underline{C}$  et  $\underline{B}$  are symmetric positive definite,

$$\underline{B} \neq \underline{C}^T \text{ !!})$$

## Changes in length

- length changes

$$\|\underline{dx}\|^2 - \|\underline{dX}\|^2 = \underline{dX} \cdot (\underline{C} - \underline{1}) \cdot \underline{dX} = \underline{dx} \cdot (\underline{1} - \underline{B}^{-1}) \cdot \underline{dx}$$

- relative elongation

$$\underline{dX} = \|\underline{dX}\| \underline{M}$$

$$\lambda(\underline{M}) = \frac{\|\underline{dx}\|}{\|\underline{dX}\|} = \sqrt{\underline{M} \cdot \underline{C} \cdot \underline{M}} = \|\underline{F} \cdot \underline{M}\| = \|\underline{U} \cdot \underline{M}\|$$

- interpretation of the components of  $\underline{C}$

$$\lambda(\underline{e}_1) = \sqrt{C_{11}} = \sqrt{F_{11}^2 + F_{21}^2 + F_{31}^2}$$

$C_{11}$  is the square of the relative elongation of the first basis vector

## Changes in angles

- variation of the angle between two material line elements

$$\underline{dX}_1 = |\underline{dX}_1| \underline{M}_1, \quad \underline{dX}_2 = |\underline{dX}_2| \underline{M}_2$$

$$\underline{dx}_1 = |\underline{dx}_1| \underline{m}_1, \quad \underline{dx}_2 = |\underline{dx}_2| \underline{m}_2$$

$$\cos \Theta = \underline{M}_1 \cdot \underline{M}_2$$

$$\cos \theta = \underline{m}_1 \cdot \underline{m}_2 = \frac{\underline{M}_1 \cdot \underline{C} \cdot \underline{M}_2}{\lambda(\underline{M}_1) \lambda(\underline{M}_2)}$$

- glide angle  $\gamma$

$$\gamma := \Theta - \theta$$

If  $\Theta = \pi/2$  (initially orthogonal directions)

$$\sin \gamma = \frac{\underline{M}_1 \cdot \underline{C} \cdot \underline{M}_2}{\lambda(\underline{M}_1) \lambda(\underline{M}_2)}$$

- interpretation of the components of  $\underline{C}$  :  $\underline{M}_1 = \underline{E}_1$  et  $\underline{M}_2 = \underline{E}_2$

$$\sin \gamma = \frac{C_{12}}{\sqrt{C_{11} C_{22}}}$$

## Rigid body motion

when the distance between any two material points does not change in the motion :

$$\begin{aligned}\forall \underline{\mathbf{X}}_1, \underline{\mathbf{X}}_2 \neq 0, \quad & \| \underline{\mathbf{x}}_1 - \underline{\mathbf{x}}_2 \| = \| \underline{\mathbf{X}}_1 - \underline{\mathbf{X}}_2 \| \\ (\underline{\mathbf{F}} \cdot (\underline{\mathbf{X}}_1 - \underline{\mathbf{X}}_2))^2 &= (\underline{\mathbf{X}}_1 - \underline{\mathbf{X}}_2) \cdot \underline{\mathbf{F}}^T \cdot \underline{\mathbf{F}} \cdot (\underline{\mathbf{X}}_1 - \underline{\mathbf{X}}_2) = (\underline{\mathbf{X}}_1 - \underline{\mathbf{X}}_2) \cdot (\underline{\mathbf{X}}_1 - \underline{\mathbf{X}}_2) \\ \implies \underline{\mathbf{F}}^T \cdot \underline{\mathbf{F}} &= \underline{\mathbf{C}} = \underline{\mathbf{1}}\end{aligned}$$

The deformation gradient is a rotation  $\underline{\mathbf{Q}}(t)$ . The corresponding motion is

$$\underline{\mathbf{x}} = \underline{\mathbf{Q}}(t) \cdot \underline{\mathbf{X}} + \underline{\mathbf{c}}(t)$$



# Strain measures

- candidates

$$\underline{\tilde{\mathbf{C}}}, \underline{\tilde{\mathbf{B}}}, \underline{\tilde{\mathbf{U}}}, \underline{\tilde{\mathbf{V}}}$$

- additional rules for defining a strain measure:

- ★ symmetric and dimensionless;
- ★ vanish for a rigid body motion and when  $\underline{\tilde{\mathbf{F}}} = \underline{\tilde{\mathbf{1}}}$ ;
- ★ the Taylor expansion around  $\underline{\tilde{\mathbf{F}}} = \underline{\tilde{\mathbf{1}}}$  is  $\frac{1}{2}(\underline{\tilde{\mathbf{H}}} + \underline{\tilde{\mathbf{H}}}^T) + o(\underline{\tilde{\mathbf{H}}})$

$$\underline{\tilde{\mathbf{H}}} = \underline{\tilde{\mathbf{F}}} - \underline{\tilde{\mathbf{1}}} = \text{Grad } \underline{\tilde{\mathbf{u}}}$$

- Green–Lagrange and Almansi tensors

$$\underline{\tilde{\mathbf{E}}} := \frac{1}{2}(\underline{\tilde{\mathbf{C}}} - \underline{\tilde{\mathbf{1}}}), \quad \underline{\tilde{\mathbf{A}}} := \frac{1}{2}(\underline{\tilde{\mathbf{1}}} - \underline{\tilde{\mathbf{B}}}^{-1})$$

$$\underline{\tilde{\mathbf{E}}} = \frac{1}{2}(\underline{\tilde{\mathbf{H}}} + \underline{\tilde{\mathbf{H}}}^T + \underline{\tilde{\mathbf{H}}}^T \cdot \underline{\tilde{\mathbf{H}}})$$

$$E_{ij} = \frac{1}{2} \left( \frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} + \frac{\partial u_k}{\partial X_i} \frac{\partial u_k}{\partial X_j} \right)$$

- **Lagrangian/Eulerian** strain measures

## Hill's strain measures

$$\mathbf{\tilde{E}}_n := \frac{1}{n}(\mathbf{\tilde{U}}^n - \mathbf{\tilde{1}}), \quad \mathbf{\tilde{A}}_n := \frac{1}{n}(\mathbf{\tilde{V}}^n - \mathbf{\tilde{1}})$$

In particular,

$$\mathbf{\tilde{E}}_2 = \mathbf{\tilde{E}}, \quad \mathbf{\tilde{A}}_{-2} = \mathbf{\tilde{A}}$$

Logarithmic strain tensor ( $n = 0$ ):

$$\mathbf{\tilde{E}}_0 := \log \mathbf{\tilde{U}}, \quad \mathbf{\tilde{A}}_0 := \log \mathbf{\tilde{V}}$$

Logarithm of a symmetric positive definite tensor:

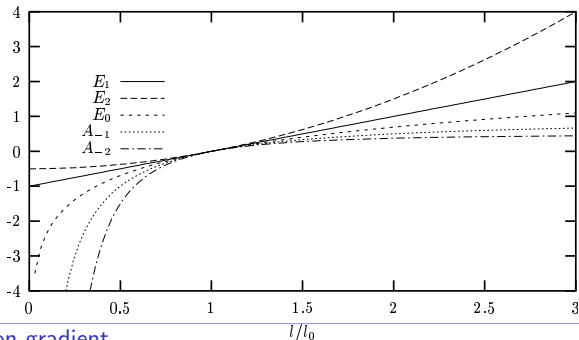
$$\mathbf{\tilde{U}} = \sum_{r=1}^3 \lambda_r \mathbf{\underline{v}}_r \otimes \mathbf{\underline{v}}_r \implies \log \mathbf{\tilde{U}} := \sum_{r=1}^3 (\log \lambda_r) \mathbf{\underline{v}}_r \otimes \mathbf{\underline{v}}_r$$

# Extensometry

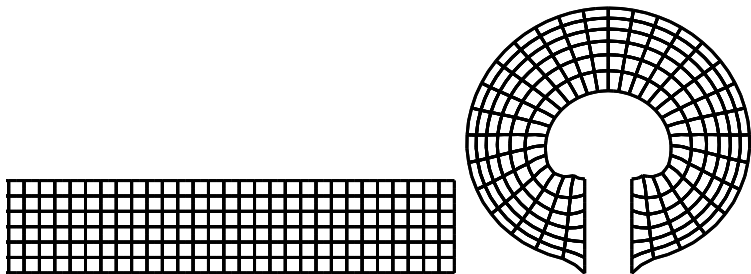
$$C = \left(\frac{l}{l_0}\right)^2, \quad B = \left(\frac{l}{l_0}\right)^2, \quad E_1 = \frac{l - l_0}{l_0}, \quad A_{-1} = \frac{l - l_0}{l}$$

$$E_2 = \frac{1}{2} \left( \left(\frac{l}{l_0}\right)^2 - 1 \right), \quad A_{-2} = \frac{1}{2} \left( 1 - \left(\frac{l_0}{l}\right)^2 \right)$$

$$E_0 = \log \frac{l}{l_0}, \quad A_0 = \log \frac{l}{l_0}$$

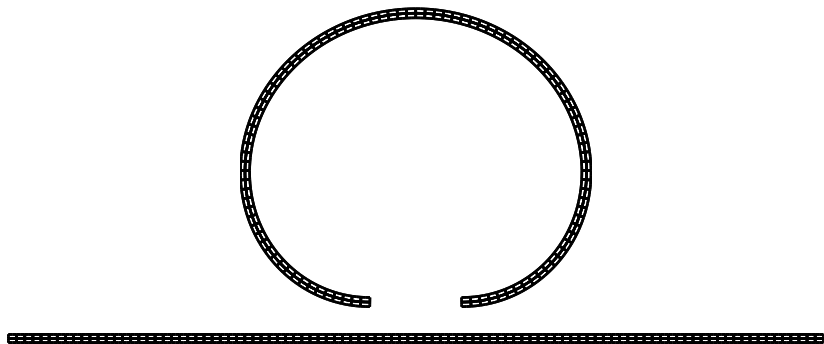


## Large strains / Large rotations



$$\tilde{\mathbf{F}} = \tilde{\mathbf{R}} \cdot \tilde{\mathbf{U}}$$

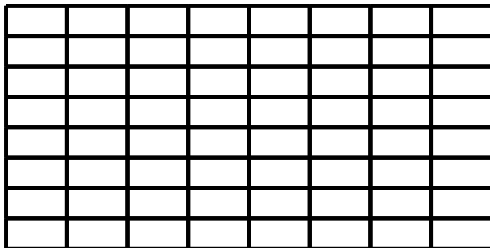
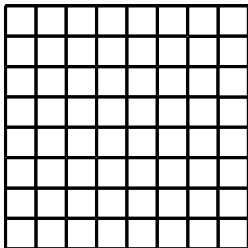
## Small strains / Large rotations



$$\tilde{\mathbf{U}} \simeq \tilde{\mathbf{1}} + \tilde{\mathbf{E}}, \quad \|\tilde{\mathbf{E}}\| \ll 1$$

for slender bodies in one or two directions (beams, plates and shells...), “large deformation” does not necessarily imply “large strain” ...

## Large strains / Small rotations



representation of small rotations

$$[\tilde{\mathbf{R}}] = \begin{bmatrix} \cos \phi & -\sin \phi & 0 \\ \sin \phi & \cos \phi & 0 \\ 0 & 0 & 1 \end{bmatrix} \simeq \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & -\phi & 0 \\ \phi & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$\tilde{\mathbf{R}} \simeq \mathbf{1} + \tilde{\boldsymbol{\omega}}$$

$\tilde{\boldsymbol{\omega}}$  skew-symmetric tensor :  $\tilde{\boldsymbol{\omega}}^T = -\tilde{\boldsymbol{\omega}}$

## Small strains / small rotations

$$\tilde{\mathbf{H}} = \text{Grad } \underline{\mathbf{u}}, \quad \tilde{\mathbf{F}} = \tilde{\mathbf{1}} + \tilde{\mathbf{H}} = \tilde{\mathbf{1}} + \tilde{\boldsymbol{\varepsilon}} + \tilde{\boldsymbol{\omega}}$$

$$\tilde{\boldsymbol{\varepsilon}} = \frac{1}{2}(\tilde{\mathbf{H}} + \tilde{\mathbf{H}}^T), \quad \tilde{\boldsymbol{\omega}} = \frac{1}{2}(\tilde{\mathbf{H}} - \tilde{\mathbf{H}}^T)$$

- context of infinitesimal deformation (with respect to a given observer) :

$$\|\tilde{\mathbf{H}} = \text{Grad } \underline{\mathbf{u}}\| \ll 1 \iff \tilde{\mathbf{F}} = \mathcal{O}(\tilde{\mathbf{1}})$$

**small deformation = small strain + small rotation**

$$\tilde{\mathbf{F}} = \tilde{\mathbf{1}} + \tilde{\boldsymbol{\varepsilon}} + \tilde{\boldsymbol{\omega}} \simeq (\tilde{\mathbf{1}} + \tilde{\boldsymbol{\omega}}) \cdot (\tilde{\mathbf{1}} + \tilde{\boldsymbol{\varepsilon}})$$

**small strain strain tensor**

$$\varepsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i})$$

**small rotation tensor**

$$\omega_{ij} := \frac{1}{2}(u_{i,j} - u_{j,i})$$

- Caution! one can always compute  $\tilde{\boldsymbol{\varepsilon}}$  but it has a physical meaning only within the infinitesimal context...

$$\tilde{\mathbf{F}} = \tilde{\mathbf{Q}} \implies \tilde{\mathbf{C}} = \tilde{\mathbf{1}}, \quad \tilde{\mathbf{E}} = 0 \quad \text{mais} \quad \tilde{\boldsymbol{\varepsilon}} = \frac{1}{2}(\tilde{\mathbf{Q}} + \tilde{\mathbf{Q}}^T) - \tilde{\mathbf{1}} \neq 0!!!$$

- $\tilde{\mathbf{C}} \simeq \tilde{\mathbf{1}} + 2\tilde{\boldsymbol{\varepsilon}}, \quad \tilde{\mathbf{E}} \simeq \tilde{\boldsymbol{\varepsilon}}$

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# Summary: Deformation of material line, surface and volume elements

material line element:  $\underline{dx} = \underline{\tilde{F}} \cdot \underline{dX}$

material surface element:  $\underline{ds} = J \underline{\tilde{F}}^{-T} \underline{dS}$

material volume element:  $dv = J dV$

## Summary: Finite deformation

$\underline{\underline{\mathbf{F}}} = \underline{\underline{\mathbf{R}}} \cdot \underline{\underline{\mathbf{U}}} = \underline{\underline{\mathbf{V}}} \cdot \underline{\underline{\mathbf{R}}}$  deformation gradient ( $\det \underline{\underline{\mathbf{F}}} > 0$ )

$\underline{\underline{\mathbf{R}}}$  polar rotation ( $\det \underline{\underline{\mathbf{R}}} = 1$ )

$\underline{\underline{\mathbf{U}}}$  right stretch tensor

$\underline{\underline{\mathbf{V}}}$  left stretch tensor

$\underline{\underline{\mathbf{C}}} := \underline{\underline{\mathbf{F}}}^T \cdot \underline{\underline{\mathbf{F}}} = \underline{\underline{\mathbf{U}}}^2$  right Cauchy–Green tensor

$\underline{\underline{\mathbf{B}}} := \underline{\underline{\mathbf{F}}} \cdot \underline{\underline{\mathbf{F}}}^T = \underline{\underline{\mathbf{V}}}^2$  left Cauchy–Green tensor

$\underline{\underline{\mathbf{E}}} := \frac{1}{2}(\underline{\underline{\mathbf{C}}} - \underline{\underline{\mathbf{1}}})$  Green–Lagrange strain measure

$\underline{\underline{\mathbf{A}}} := \frac{1}{2}(\underline{\underline{\mathbf{1}}} - \underline{\underline{\mathbf{B}}}^{-1})$  Almansi strain measure

# Summary: Infinitesimal deformation

$$\underline{\mathbf{H}} = \underline{\boldsymbol{\varepsilon}} + \underline{\boldsymbol{\omega}} = \text{Grad } \underline{\mathbf{u}} \quad \text{displacement gradient}$$

$$\underline{\boldsymbol{\varepsilon}} = \frac{1}{2}(\text{Grad } \underline{\mathbf{u}} + (\text{Grad } \underline{\mathbf{u}})^T) \quad \text{small strain tensor}$$

$$\underline{\boldsymbol{\omega}} = \frac{1}{2}(\text{Grad } \underline{\mathbf{u}} - (\text{Grad } \underline{\mathbf{u}})^T) \quad \text{small rotation tensor}$$

$$\underline{\mathbf{F}} = \underline{\mathbf{1}} + \underline{\boldsymbol{\varepsilon}} + \underline{\boldsymbol{\omega}} \simeq (\underline{\mathbf{1}} + \underline{\boldsymbol{\varepsilon}}) \cdot (\underline{\mathbf{1}} + \underline{\boldsymbol{\omega}}), \quad \underline{\mathbf{C}} \simeq \underline{\mathbf{1}} + 2\underline{\boldsymbol{\varepsilon}} \simeq \underline{\mathbf{B}}, \quad \underline{\mathbf{E}} \simeq \underline{\boldsymbol{\varepsilon}}$$

$$\frac{|\underline{\mathbf{d}}\mathbf{x}| - |\underline{\mathbf{d}}\mathbf{X}|}{|\underline{\mathbf{d}}\mathbf{X}|} \simeq \underline{\mathbf{M}} \cdot \underline{\boldsymbol{\varepsilon}} \cdot \underline{\mathbf{M}} \quad \text{infinitesimal elongation}$$

$$\frac{dv - dV}{dV} \simeq \text{div } \underline{\mathbf{u}} = \text{trace } \underline{\boldsymbol{\varepsilon}} \quad \text{infinitesimal volume change}$$

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# Velocity field

- Notations

$$\dot{\underline{\mathbf{x}}} = \frac{d\underline{\mathbf{x}}}{dt} = \frac{\partial \Phi}{\partial t}(\underline{\mathbf{X}}, t) = \underline{\mathbf{V}}(\underline{\mathbf{X}}, t)$$

- Lagrangian/Eulerian representations

$$\underline{\mathbf{v}}(\underline{\mathbf{x}}, t) := \underline{\mathbf{V}}(\Phi^{-1}(\underline{\mathbf{x}}, t), t)$$

more generally

$$f(\underline{\mathbf{x}}, t) := F(\underline{\mathbf{X}}, t), \quad \text{avec} \quad \underline{\mathbf{x}} = \Phi(\underline{\mathbf{X}}, t)$$

- convective time derivative

$$\begin{aligned} \dot{F}(\underline{\mathbf{X}}, t) &:= \frac{d}{dt} F(\underline{\mathbf{X}}, t) = \frac{\partial F}{\partial t}(\underline{\mathbf{X}}, t) \\ &= \frac{d}{dt} f(\underline{\mathbf{x}}, t) = \frac{\partial f}{\partial t}(\underline{\mathbf{x}}, t) + \frac{\partial f}{\partial \underline{\mathbf{x}}} \cdot \underline{\mathbf{v}}(\underline{\mathbf{x}}, t) = \dot{f}(\underline{\mathbf{x}}, t) \end{aligned}$$

# Velocity gradient field

- instantaneous evolution of a material line element

$$\underline{dx} = \underline{\dot{\mathbf{F}}} \cdot \underline{d\mathbf{X}}$$

$$\overbrace{\underline{dx}}^{\bullet} = \underline{\mathbf{L}} \cdot \underline{dx}, \quad \text{with} \quad \underline{\mathbf{L}} = \underline{\dot{\mathbf{F}}} \cdot \underline{\mathbf{F}}^{-1}$$

- velocity gradient tensor

$$\begin{aligned} \underline{\dot{\mathbf{F}}} &= \frac{\partial^2 \Phi}{\partial t \partial \underline{\mathbf{X}}}(\underline{\mathbf{X}}, t) = \frac{\partial^2 \Phi}{\partial \underline{\mathbf{X}} \partial t}(\underline{\mathbf{X}}, t) \\ &= \text{Grad } \underline{\mathbf{V}}(\underline{\mathbf{X}}, t) = (\text{grad } \underline{\mathbf{v}}(\underline{\mathbf{x}}, t)) \cdot \underline{\mathbf{F}} \\ \underline{\mathbf{L}}(\underline{\mathbf{x}}, t) &= \text{grad } \underline{\mathbf{v}}(\underline{\mathbf{x}}, t) = \underline{\dot{\mathbf{F}}} \cdot \underline{\mathbf{F}}^{-1} \end{aligned}$$

## Strain rate tensor

- instantaneous evolution of the scalar product of two material line elements  
on the one hand...

$$\overbrace{\underline{\mathbf{dx}}_1 \cdot \underline{\mathbf{dx}}_2}^{\bullet} = \underline{\mathbf{dx}}_1 \cdot \underline{\mathbf{L}}^T \cdot \underline{\mathbf{dx}}_2 + \underline{\mathbf{dx}}_1 \cdot \underline{\mathbf{L}} \cdot \underline{\mathbf{dx}}_2 = 2 \underline{\mathbf{dx}}_1 \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{dx}}_2$$

... and on the other hand

$$\overbrace{\underline{\mathbf{dx}}_1 \cdot \underline{\mathbf{dx}}_2}^{\bullet} = \overbrace{\underline{\mathbf{dX}}_1 \cdot \underline{\mathbf{C}} \cdot \underline{\mathbf{dX}}_2}^{\bullet} = \underline{\mathbf{dX}}_1 \cdot \underline{\dot{\mathbf{C}}} \cdot \underline{\mathbf{dX}}_2 = 2 \underline{\mathbf{dX}}_1 \cdot \underline{\dot{\mathbf{E}}} \cdot \underline{\mathbf{dX}}_2$$

hence ...

$$\underline{\dot{\mathbf{E}}} = \frac{1}{2} \underline{\dot{\mathbf{C}}} = \underline{\mathbf{F}}^T \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{F}}, \quad \underline{\mathbf{D}} := \frac{1}{2} (\underline{\mathbf{L}} + \underline{\mathbf{L}}^T)$$

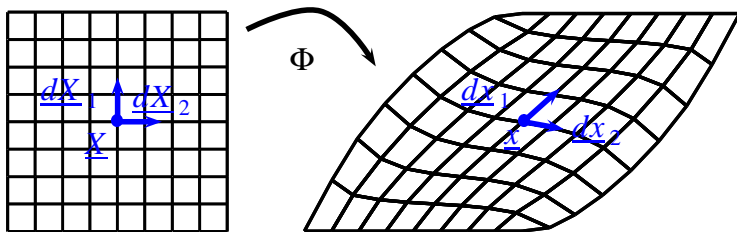
strain rate tensor

- elongation rate :

$$\underline{\mathbf{dx}} = \|\underline{\mathbf{dx}}\| \underline{\mathbf{m}}, \quad \underline{\mathbf{m}} \text{ unit vector} \qquad \frac{\dot{\lambda}}{\lambda} = \overbrace{\frac{\|\underline{\mathbf{dx}}\|}{\|\underline{\mathbf{dx}}\|}}^{\bullet} = \underline{\mathbf{m}} \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{m}}$$



## Slip rate



- glide angle :  $\gamma = \Theta - \theta$

$$\dot{\gamma} = -\dot{\theta}$$

$$\dot{\underline{\mathbf{dx}}_1 \cdot \underline{\mathbf{dx}}_2} = \overbrace{\|\underline{\mathbf{dx}}_1\| \|\underline{\mathbf{dx}}_2\| \cos \theta}^{\dot{\gamma}} = 2 \underline{\mathbf{dx}}_1 \cdot \tilde{\underline{\mathbf{D}}} \cdot \underline{\mathbf{dx}}_2$$

If  $\theta = \frac{\pi}{2}$  at time  $t$ ,

$$\dot{\gamma} = 2 \underline{\mathbf{m}}_1 \cdot \tilde{\underline{\mathbf{D}}} \cdot \underline{\mathbf{m}}_2$$

$$\circ \underline{\mathbf{m}}_1 = \underline{\mathbf{dx}}_1 / \|\underline{\mathbf{dx}}_1\|, \quad \underline{\mathbf{m}}_2 = \underline{\mathbf{dx}}_2 / \|\underline{\mathbf{dx}}_2\|$$

- particular case,  $\underline{\mathbf{m}}_1 = \underline{\mathbf{e}}_1, \quad \underline{\mathbf{m}}_2 = \underline{\mathbf{e}}_2 \implies \dot{\gamma} = 2D_{12}$

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# Spin tensor

- instantaneous evolution of the orientation of a material line element  $\underline{\mathbf{m}} = \underline{\mathbf{dx}} / \|\underline{\mathbf{dx}}\|$

$$\dot{\underline{\mathbf{m}}} = \underline{\mathbf{L}} \cdot \underline{\mathbf{m}} - (\underline{\mathbf{m}} \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{m}}) \underline{\mathbf{m}}$$

- If  $\underline{\mathbf{m}}$  is parallel to a principal vector of  $\underline{\mathbf{D}}$

$$\underline{\mathbf{W}} := \underline{\mathbf{L}} - \underline{\mathbf{D}} = \frac{1}{2}(\underline{\mathbf{L}} - \underline{\mathbf{L}}^T)$$

$$\dot{\underline{\mathbf{m}}} = \underline{\mathbf{W}} \cdot \underline{\mathbf{m}}$$

spin tensor

- Consequence : *The orthonormal triad of material vectors coinciding at time  $t$  with the triad of unit eigenvectors of  $\underline{\mathbf{D}}$  transforms like a rigid body with the rotation rate  $\underline{\mathbf{W}}$  at time  $t$*
- Caution! The triad of eigenvectors of  $\underline{\mathbf{D}}$  generally does not rotate at the rate  $\underline{\mathbf{W}}$ ... (counterexample: simple glide)

# Decomposition of the velocity gradient tensor

- strain rate + spin

$$\underline{\underline{\mathbf{L}}} = \underline{\underline{\mathbf{D}}} + \underline{\underline{\mathbf{W}}}$$

symmetric and skew-symmetric parts

- polar decomposition

$$\underline{\underline{\mathbf{F}}} = \underline{\underline{\mathbf{R}}} \cdot \underline{\underline{\mathbf{U}}}$$

$$\underline{\underline{\mathbf{L}}} = \dot{\underline{\underline{\mathbf{F}}}} \cdot \underline{\underline{\mathbf{F}}}^{-1} = \dot{\underline{\underline{\mathbf{R}}}} \cdot \underline{\underline{\mathbf{R}}}^T + \underline{\underline{\mathbf{R}}} \cdot \dot{\underline{\underline{\mathbf{U}}}} \cdot \underline{\underline{\mathbf{U}}}^{-1} \cdot \underline{\underline{\mathbf{R}}}^T$$

Caution! the last term is generally not symmetric... In general,

$$\underline{\underline{\mathbf{W}}} \neq \dot{\underline{\underline{\mathbf{R}}}} \cdot \underline{\underline{\mathbf{R}}}^T$$

- spin vector

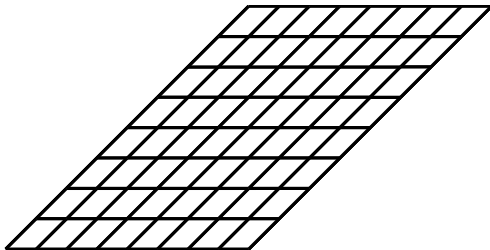
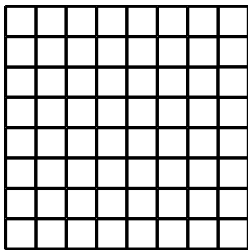
$$\forall \underline{\underline{\mathbf{y}}}, \quad \underline{\underline{\mathbf{W}}} \cdot \underline{\underline{\mathbf{y}}} = \underline{\underline{\mathbf{W}}}^{\times} \wedge \underline{\underline{\mathbf{y}}}$$

$$\left\{ \begin{array}{l} \underline{\underline{\mathbf{W}}}^{\times}_1 = -W_{23} = \frac{1}{2} \left( \frac{\partial v_3}{\partial x_2} - \frac{\partial v_2}{\partial x_3} \right) \\ \underline{\underline{\mathbf{W}}}^{\times}_2 = -W_{31} = \frac{1}{2} \left( \frac{\partial v_1}{\partial x_3} - \frac{\partial v_3}{\partial x_1} \right) \\ \underline{\underline{\mathbf{W}}}^{\times}_3 = -W_{12} = \frac{1}{2} \left( \frac{\partial v_2}{\partial x_1} - \frac{\partial v_1}{\partial x_2} \right) \end{array} \right. , \quad \underline{\underline{\mathbf{W}}}^{\times} = \frac{1}{2} \text{rot } \underline{\underline{\mathbf{v}}}$$

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## Simple glide

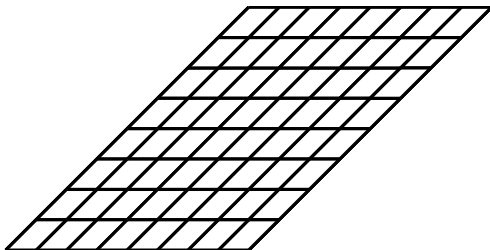
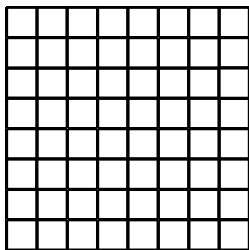


$$[\tilde{\mathbf{L}}] = \begin{bmatrix} 0 & \dot{\gamma} & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$[\tilde{\mathbf{D}}] = \begin{bmatrix} 0 & \frac{\dot{\gamma}}{2} & 0 \\ \frac{\dot{\gamma}}{2} & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$[\tilde{\mathbf{W}}] = \begin{bmatrix} 0 & \frac{\dot{\gamma}}{2} & 0 \\ -\frac{\dot{\gamma}}{2} & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

## Simple glide



$$[\mathbf{W}] = \begin{bmatrix} 0 & \frac{\dot{\gamma}}{2} & 0 \\ -\frac{\dot{\gamma}}{2} & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, \quad [\mathbf{R}] = \begin{bmatrix} \frac{1}{\sqrt{1+(\gamma/2)^2}} & \frac{\gamma}{2\sqrt{1+(\gamma/2)^2}} & 0 \\ \frac{-\gamma}{2\sqrt{1+(\gamma/2)^2}} & \frac{1}{\sqrt{1+(\gamma/2)^2}} & 0 \\ 0 & 0 & 1 \end{bmatrix}$$
$$\dot{\theta}_W = -\frac{\dot{\gamma}}{2}, \quad \tan \theta_R = -\frac{\gamma}{2}, \quad \dot{\theta}_R = -\frac{\dot{\gamma}}{2} \frac{1}{1 + \gamma^2/4}$$

# Single vortex





# Single vortex

- kinematicse

$$\underline{\mathbf{v}}(r, \theta, z, t) = \frac{\Gamma}{2\pi r} \underline{\mathbf{e}}_{\theta}$$

Current lines are circles around the vortex center

- velocity gradient

$$\underline{\mathbf{L}} = -\frac{\Gamma}{2\pi r^2} (\underline{\mathbf{e}}_r \otimes \underline{\mathbf{e}}_{\theta} + \underline{\mathbf{e}}_{\theta} \otimes \underline{\mathbf{e}}_r)$$

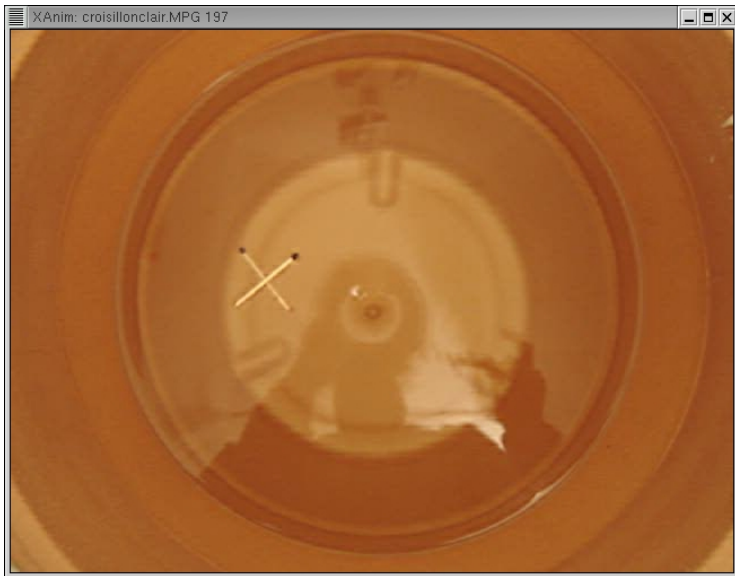
- irrotational flow

$$\underline{\mathbf{W}} = 0$$

- circulation of  $\underline{\mathbf{v}}$  around O

$$\oint \underline{\mathbf{v}} \cdot \underline{\mathbf{e}}_{\theta} r d\theta = \Gamma$$

# Vorticitymeter



# Vorticity (1)

- unit directions characterising the cross  $\underline{\mathbf{m}}_1$  et  $\underline{\mathbf{m}}_2$

$$\dot{\underline{\mathbf{m}}}_1 = \underline{\mathbf{L}} \cdot \underline{\mathbf{m}}_1 - (\underline{\mathbf{m}}_1 \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{m}}_1) \underline{\mathbf{m}}_1$$

$$\dot{\underline{\mathbf{m}}}_2 = \underline{\mathbf{L}} \cdot \underline{\mathbf{m}}_2 - (\underline{\mathbf{m}}_2 \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{m}}_2) \underline{\mathbf{m}}_2$$

- Evolution of the angle between one match and a fixed direction in space  $\underline{\mathbf{a}}$

$$-\sin \varphi_1 \dot{\varphi}_1 = \dot{\underline{\mathbf{m}}}_1 \cdot \underline{\mathbf{a}} = \underline{\mathbf{a}} \cdot \underline{\mathbf{L}} \cdot \underline{\mathbf{m}}_1 - (\underline{\mathbf{m}}_1 \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{m}}_1) \underline{\mathbf{m}}_1 \cdot \underline{\mathbf{a}}$$

The choice of  $\underline{\mathbf{a}}$  does not matter if we are interested in  $\dot{\varphi}$ .  
Take

$$\varphi_1 = (\underline{\mathbf{a}} = \underline{\mathbf{m}}_2, \underline{\mathbf{m}}_1) = -\frac{\pi}{2} \implies \dot{\varphi}_1 = \underline{\mathbf{m}}_2 \cdot \underline{\mathbf{L}} \cdot \underline{\mathbf{m}}_1$$

$$\varphi_2 = (\underline{\mathbf{a}} = \underline{\mathbf{m}}_1, \underline{\mathbf{m}}_2) = \frac{\pi}{2} \implies \dot{\varphi}_2 = \underline{\mathbf{m}}_1 \cdot \underline{\mathbf{L}} \cdot \underline{\mathbf{m}}_2$$

## Vorticitymeter (2)

- For a rigid cross ( $\underline{\mathbf{m}}_1 \cdot \underline{\mathbf{m}}_2 = 0$  at each time), the spin of the cross is the mean value of the spin of the matches :

$$\begin{aligned}\dot{\varphi} &= \frac{\dot{\varphi}_1 + \dot{\varphi}_2}{2} = \underline{\mathbf{m}}_2 \cdot \underline{\tilde{\mathbf{W}}} \cdot \underline{\mathbf{m}}_1 \\ &= \underline{\mathbf{m}}_2 \cdot (\underline{\tilde{\mathbf{W}}}^\times \wedge \underline{\mathbf{m}}_1) = \underline{\tilde{\mathbf{W}}}^\times \wedge (\underline{\mathbf{m}}_1 \wedge \underline{\mathbf{m}}_2) = \underline{\tilde{\mathbf{W}}}^\times \cdot \underline{\mathbf{e}}_z\end{aligned}$$

- The spin of the rigid cross is exactly that of the spin tensor of the fluid. The vorticitymeter can be used to measure  $\underline{\tilde{\mathbf{W}}}$ .
- For a simple vortex,  $\underline{\tilde{\mathbf{W}}} = 0$ . The cross does not rotate...

# Plan

- 1 Strain field measurements
- 2 Material placement
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  - Simple extension and simple glide
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- 4 Velocity gradient tensor
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# Analogy $\underline{\mathbf{D}} \longleftrightarrow \underline{\boldsymbol{\varepsilon}}$

	strain rate $\underline{\mathbf{D}}$ (general case)	small strain $\underline{\boldsymbol{\varepsilon}}$ (infinitesimal context)
symmetric gradient operator	$\underline{\mathbf{D}} = \frac{1}{2}(\text{grad } \underline{\mathbf{v}} + \text{grad } \underline{\mathbf{v}})$ $D_{ij} = \frac{1}{2}(v_{i,j} + v_{j,i})$	$\underline{\boldsymbol{\varepsilon}} = \frac{1}{2}(\text{Grad } \underline{\mathbf{u}} + \text{Grad } \underline{\mathbf{u}})$ $\varepsilon_{ij} = \frac{1}{2}(u_{i,j} + u_{j,i})$
volume  change	$\frac{\dot{dV}}{dV} = \text{div } \underline{\mathbf{v}} = \text{trace } \underline{\mathbf{D}}$	$\frac{dV - dV}{dV} \simeq \text{Div } \underline{\mathbf{u}} = \text{trace } \underline{\boldsymbol{\varepsilon}}$
relative elongation	$\frac{\dot{\lambda}}{\lambda} = \underline{\mathbf{m}} \cdot \underline{\mathbf{D}} \cdot \underline{\mathbf{m}}$	$\lambda - 1 \simeq \underline{\mathbf{M}} \cdot \underline{\boldsymbol{\varepsilon}} \cdot \underline{\mathbf{M}} \simeq \frac{\lambda-1}{\lambda}$

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# Principle of virtual power

- Power of external forces acting on material domain  $\mathcal{D} \subset \Omega_t$

$$\mathcal{P}^c(\underline{\mathbf{v}}^*) + \mathcal{P}^e(\underline{\mathbf{v}}^*) = \int_{\partial\mathcal{D}} \underline{\mathbf{t}} \cdot \underline{\mathbf{v}}^* ds + \int_{\mathcal{D}} \rho \underline{\mathbf{f}} \cdot \underline{\mathbf{v}}^* dv$$

- Power of acceleration forces

$$\mathcal{P}^a(\underline{\mathbf{v}}^*) := \int_{\mathcal{D}} \rho \underline{\mathbf{a}} \cdot \underline{\mathbf{v}}^* dv$$

- Power of internal forces, stress tensor

$$\mathcal{P}^i(\underline{\mathbf{v}}^*) := - \int_{\mathcal{D}} p^{(i)} dv, \quad p^{(i)} = \underline{\boldsymbol{\sigma}} : \underline{\mathbf{D}}^* \sim \text{MPa.s}^{-1} = \text{Wm}^{-3}$$

- Principle of virtual power,  $\forall \underline{\mathbf{v}}^*$  (regular),  $\forall \mathcal{D} \subset \Omega_t$

$$\begin{aligned} \mathcal{P}^c(\underline{\mathbf{v}}^*) + \mathcal{P}^e(\underline{\mathbf{v}}^*) + \mathcal{P}^i(\underline{\mathbf{v}}^*) &= \mathcal{P}^a(\underline{\mathbf{v}}^*) \\ - \int_{\mathcal{D}} \underline{\boldsymbol{\sigma}} : \underline{\mathbf{D}}^* dv + \int_{\partial\mathcal{D}} \underline{\mathbf{t}} \cdot \underline{\mathbf{v}}^* ds + \int_{\mathcal{D}} \rho \underline{\mathbf{f}} \cdot \underline{\mathbf{v}}^* dv &= \int_{\mathcal{D}} \rho \underline{\mathbf{a}} \cdot \underline{\mathbf{v}}^* dv \end{aligned}$$

# Principle of virtual power

- Principle of virtual power (regular case, no shock wave)

$$\mathcal{P}^i(\underline{\mathbf{v}}^*) + \mathcal{P}^c(\underline{\mathbf{v}}^*) + \mathcal{P}^e(\underline{\mathbf{v}}^*) = \mathcal{P}^a(\underline{\mathbf{v}}^*)$$

$$- \int_{\mathcal{D}} \underline{\boldsymbol{\sigma}} : \underline{\mathbf{D}}^* dv + \int_{\partial\mathcal{D}} \underline{\mathbf{t}} \cdot \underline{\mathbf{v}}^* ds + \int_{\mathcal{D}} \rho \underline{\mathbf{f}} \cdot \underline{\mathbf{v}}^* dv = \int_{\mathcal{D}} \rho \underline{\mathbf{a}} \cdot \underline{\mathbf{v}}^* dv$$

- equivalent to the field equations (balance of momentum and moment of momentum)

$$\operatorname{div} \underline{\boldsymbol{\sigma}} + \rho \underline{\mathbf{f}} = \rho \underline{\mathbf{a}}, \quad \forall \underline{\mathbf{x}} \in \Omega_t$$

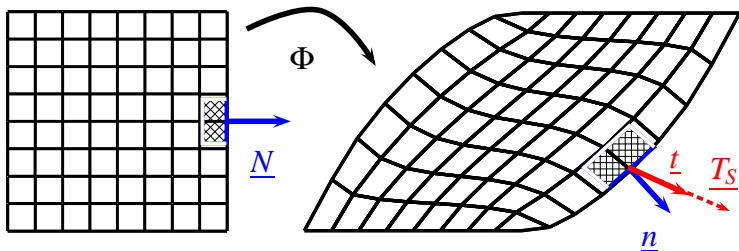
$$\underline{\boldsymbol{\sigma}}^T = \underline{\boldsymbol{\sigma}}$$

$$\underline{\mathbf{t}} = \underline{\boldsymbol{\sigma}} \cdot \underline{\mathbf{n}}, \quad \forall \underline{\mathbf{x}} \in \partial\Omega_t$$

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## Nominal stress tensor



- Lagrangean version of the field equations

$$\int_{\mathcal{D}} \rho \underline{\mathbf{a}} \, dv = \int_{\partial \mathcal{D}} \underline{\boldsymbol{\sigma}} \cdot \underline{\mathbf{n}} \, ds + \int_{\mathcal{D}} \rho \underline{\mathbf{f}} \, dv$$

$$\int_{\mathcal{D}_0} \rho_0 \underline{\mathbf{A}} \, dV = \int_{\partial \mathcal{D}_0} \underline{\mathbf{S}} \cdot \underline{\mathbf{N}} \, dS + \int_{\mathcal{D}_0} \rho_0 \underline{\mathbf{F}} \, dV$$

- nominal stress tensor or Boussinesq stress tensor  $\underline{\mathbf{S}}$

$$\underline{\mathbf{n}} \, ds = J \, \underline{\mathbf{F}}^{-T} \cdot \underline{\mathbf{N}} \, dS$$

$$\underline{\mathbf{t}} \, ds = \underline{\mathbf{T}}_S \, dS = \underline{\mathbf{S}} \cdot \underline{\mathbf{N}} \, dS, \quad \text{avec} \quad \underline{\mathbf{S}} := J \underline{\boldsymbol{\sigma}} \cdot \underline{\mathbf{F}}^{-T}$$

## Piola–Kirchhoff stress tensor

- power density of internal forces

$$\int_{\mathcal{D}} \underline{\underline{\sigma}} : \underline{\underline{D}} \, dv = \int_{\mathcal{D}} \underline{\underline{\Pi}} : \underline{\underline{\dot{E}}} \, dV$$

$$\underline{\underline{\Pi}} = J \underline{\underline{F}}^{-1} \cdot \underline{\underline{\sigma}} \cdot \underline{\underline{F}}^{-T} = \underline{\underline{F}}^{-1} \cdot \underline{\underline{S}}$$

Piola–Kirchhoff stress tensor  $\underline{\underline{\Pi}}$

- mass density of power of internal forces

$$\frac{\underline{\underline{\sigma}} : \underline{\underline{D}}}{\rho} = \frac{\underline{\underline{\Pi}} : \underline{\underline{\dot{E}}}}{\rho_0}$$

conjugate stress–strain measures

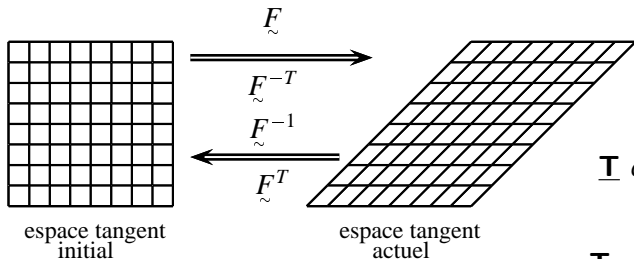
- transport of the traction vector

$$\underline{\underline{T}} \, dS := \underline{\underline{F}}^{-1} \cdot \underline{\underline{t}} \, ds = \underline{\underline{F}}^{-1} \cdot \underline{\underline{T}}_S \, dS = \underline{\underline{\Pi}} \cdot \underline{\underline{N}} \, dS$$

## Remark on transport rules

$$\underline{dx} = \underline{\tilde{F}} \cdot \underline{dX}$$

$$\underline{ds} = J \underline{\tilde{F}}^{-T} \cdot \underline{dS}$$



$$\underline{\mathbf{T}} dS = \underline{\tilde{F}}^{-1} \cdot \underline{\mathbf{t}} ds$$

$$\begin{aligned} \underline{\mathbf{T}}_M dS &= \underline{\tilde{F}}^T \cdot \underline{\mathbf{t}} ds \\ &= \underline{\tilde{\mathbf{C}}} \cdot \underline{\tilde{\mathbf{\Pi}}} \cdot \underline{\mathbf{N}} dS \end{aligned}$$

Mandel stress tensor